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General information

These application notes provide a simplified procedure for sizing control valves at standard service conditions.

The data required to size a valve, such as the nominal size, nominal pressures and K_{VS} , is contained in the SAMSON data sheets for self-operated regulators and control valves.

Self-operated regulators and control valves can be sized accurately using the DIN IEC 534 procedure. For most applications, however, the following sizing equations formulated in the VDI/VDE Guideline 2173 provide a sufficient degree of accuracy.

In order to calculate the valve flow coefficient K_V , the operating data specified in the figure to the right is required.

Typical sizing coefficients

Explanatory notes

Control valves and self-operated regulators

Rated travel · The rated travel H_{100} is the amount of movement of the valve closure member from the closed position to the designated full open position published by the manufacturer for each control valve series.

K_V · The valve flow coefficient K_V is defined as the number of cubic meters per hour (volume flow rate \dot{V}) of 5 to 30 °C water that will flow through a control valve at a specified travel H with a differential pressure ($\Delta p = p_1 - p_2$) of 10^5 Pa (1 bar) across it.

K_{VS} · The K_{VS} value is the expected flow coefficient K_V of the valve at rated travel H_{100} indicated and published by the manufacturer for each valve type (series).

K_{V100} · The K_{V100} value is the effective (actual) flow coefficient K_V of the valve at rated travel H_{100} . It must not deviate by more than $\pm 10\%$ from the indicated K_{VS} value.

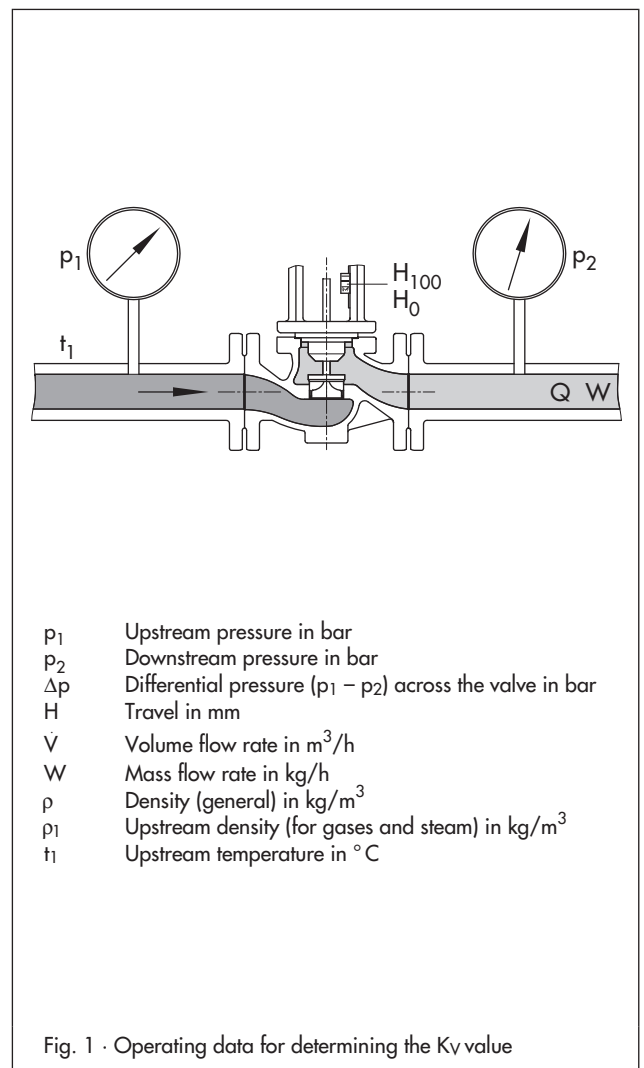
Self-operated regulators

Self-operated flow regulators can only be sized if the upstream pressure p_1 is constant.

$$\text{Safety factor } S = \frac{K_{VS}}{K_V} \quad \begin{array}{ll} K_{VS}: & \text{Indicated } K_{VS} \\ K_V: & \text{Calculated } K_V \end{array}$$

For self-operated regulators: $S \approx 1.3$ to 5

To ensure the proper operation of a self-operated regulator, the **kinematic viscosity** ν if the medium to be controlled must not exceed $1 \cdot 10^{-4} \text{ m}^2/\text{s} = 100 \text{ cSt}$.



Conversion of the maximum volume flow rate for liquids with different densities

$$\dot{V}_B = \dot{V}_A \cdot \left(\sqrt{\frac{\rho_A}{\rho_B}} \right) \quad \begin{array}{ll} \rho & \text{Density in } \text{kg}/\text{m}^3 \\ \dot{V} & \text{Volume flow rate in } \text{m}^3/\text{h} \end{array}$$

Equation (1) generally applies to liquids:

$$\dot{V} = K_V \cdot \sqrt{\frac{1000 \cdot \Delta p}{\rho}} \quad (1) \quad \text{This equation contains the dimensional factor 1.}$$

Diagram 1 shows the relationship between \dot{V} , K_V and Δp for liquids that have a density of $\rho = 1000 \text{ kg/m}^3$ at a temperature of $t = 20 \text{ }^\circ\text{C}$.

Flow regulators · Differential pressure across the valve

Calculate the minimum required differential pressure across the valve using Equation (2):

$$\Delta p_{\min} = \Delta p_{\text{restriction}} + \left(\frac{\dot{V}}{K_{VS}} \right)^2 \quad (2)$$

Symbols and units

p_1	} Absolute pressure
p_2	
Δp	In bar
ρ	Density in kg/m^3
\dot{V}	Volume flow rate in m^3/h
K_V	K_V in m^3/h
Δp_{\min}	Minimum differential pressure across the valve in bar
$\Delta p_{\text{restriction}}$	Differential pressure in bar created deliberately across the restriction in the flow path for flow measurement purposes
K_{VS}	Indicated valve sizing coefficient in m^3/h

Sizing sample problem 1

Determine: Min. differential pressure Δp across fully open valve
 Given: Type 42-36 Self-operated Flow Regulator, diff. pressure across restriction 0.2 bar, DN 40, $K_{VS} 20$, volume flow rate
 Solution: Calculate Δp_{\min} using Equation (2):

$$\Delta p_{\min} = \Delta p_{\text{restriction}} + \left(\frac{\dot{V}}{K_{VS}} \right)^2 \quad (2)$$

$\Delta p_{\text{restriction}} = 0.2$ to 0.5 bar depending on regulator version

$$\begin{aligned} K_{VS} &= 20 \text{ m}^3/\text{h} \\ \Delta p_{\text{restriction}} &= 0.2 \text{ bar} \\ \dot{V} &= 10 \text{ m}^3/\text{h} \\ \Delta p_{\min} &= 0.2 + \left(\frac{10}{20} \right)^2 = \mathbf{0.45 \text{ bar}} \end{aligned}$$

Sizing sample problem 2

Determine: Volume flow rate of acetone (m^3/h) with valve fully open
 Given: Type 3241-1 Pneumatic Control Valve, DN 40, differential pressure $\Delta p = p_1 - p_2$ · Density of acetone in kg/m^3

Solution: Calculate \dot{V} using Equation (1):

$$\dot{V} = K_{VS} \cdot \sqrt{\frac{1000 \cdot \Delta p}{\rho}} \quad (1)$$

$$\begin{aligned} K_{VS} &= 25 \text{ m}^3/\text{h}^{1)} \\ \Delta p &= p_1 - p_2 = 0.5 \text{ bar} \\ \rho &= 800 \text{ kg/m}^3 \\ \dot{V} &= 25 \cdot \sqrt{\frac{1000 \cdot 0.5}{800}} = \mathbf{19.76 \text{ m}^3/\text{h}} \end{aligned}$$

1) K_{VS} and hence also \dot{V} , the calculated volume flow rate, have a permissible tolerance of $\pm 10 \%$.

Sizing sample problem 3

Determine: Type ... Pressure Reducing Valve, K_{VS} , nominal valve size
 Given: Volume flow rate of water · Differential pressure · Density of water ρ in kg/m^3
 Solution: Calculate the K_V value using Equation (3) derived from Equation (1):

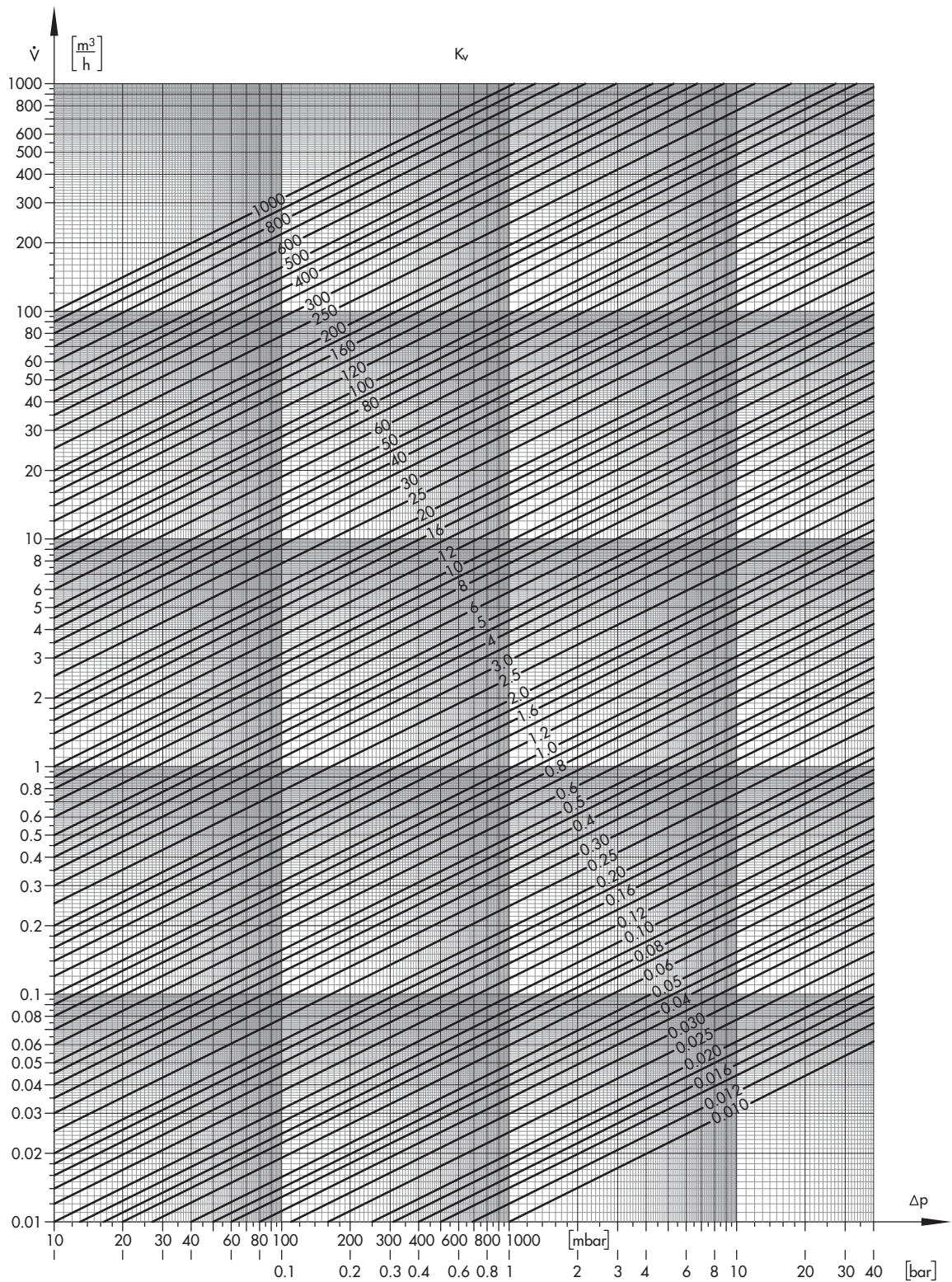
$$K_V = \dot{V} \cdot \sqrt{\frac{\rho}{1000 \cdot \Delta p}} \quad (3)$$

Determine the K_{VS} value from the following equation using the calculated K_V value. In general: $K_{VS} \approx 1.3 \cdot K_V$
 Solution provided by Diagram 1:

For $\Delta p = 2.1$ bar and $\dot{V} = 12 \text{ m}^3/\text{h}$, a K_V value of approximately $8.2 \text{ m}^3/\text{h}$ is indicated in Diagram 1.
 The value for the flow velocity in the pipe can be found in Diagram 4 on page 11 using $\dot{V} = 12 \text{ m}^3/\text{h}$ and DN 40:
 $w_{\text{pipe}} \approx 2.8 \text{ m/s}$

$$\begin{aligned} \dot{V} &= 12 \text{ m}^3/\text{h} \\ \Delta p &= p_1 - p_2 = 2.1 \text{ bar} \\ \rho &= 1000 \text{ kg/m}^3 \\ K_V &= 12 \cdot \sqrt{\frac{1000}{1000 \cdot 2.1}} = \mathbf{8.2 \text{ m}^3/\text{h}} \\ K_{VS} &= 1.3 \cdot K_V = 1.3 \cdot 8.2 = \mathbf{10.7 \text{ m}^3/\text{h}} \\ \text{Selection: Type 41-23, DN 40, } K_{VS} &= 20 \end{aligned}$$

Determining S, the safety factor:
 $S = \frac{K_{VS}}{K_V} = \frac{20}{8.2} \approx 2.4$



\dot{V} in m^3/h

K_V in m^3/h

Δp in bar

$$\dot{V} = K_V \cdot \sqrt{\frac{1000 \cdot \Delta p}{\rho}}$$

Diagram 1 · Volume flow rate diagram for water with $\rho = 1000 \text{ kg}/\text{m}^3$, $t = 20 \text{ }^\circ\text{C}$

Sizing sample problem 4

Determine: Differential pressure $\Delta p = p_1 - p_2$ with valve fully open
 Given: Type 4 Self-operated Temperature Regulator with DN 50,
 \dot{V} , volume flow rate of water,
 ρ , density of water in kg/m^3

$K_{VS} = 32 \text{ m}^3/\text{h}$
 $\dot{V} = 10 \text{ m}^3/\text{h}$
 $\rho = 1000 \text{ kg/m}^3$

Solution: Determine the differential pressure using Equation (4)
 derived from Equation (1):

$$\Delta p = \left(\frac{10}{32}\right)^2 \cdot \frac{1000}{1000} = \mathbf{0.1 \text{ bar}}$$

$$\Delta p = \left(\frac{\dot{V}}{K_{VS}}\right)^2 \cdot \frac{\rho}{1000} \quad (4)$$

Solution provided by Diagram 1:

For $\dot{V} = 10 \text{ m}^3/\text{h}$ and $K_{VS} 32$, a differential pressure of
 $\Delta p \approx 0.1 \text{ bar}$ is obtained from Diagram 1.

A modified version of Equation (8) applies to steam:

$$W = K_V \cdot m \cdot Z \quad (5)$$

- Z** The dimensionless compressibility factor Z is defined as follows: $Z = 14.2 \cdot \sqrt{p_1 \cdot \rho_1}$. Determine Z from Table 2 using the upstream pressure p_1 and differentiating between saturated steam and superheated steam.
- m** Determine the dimensionless head loss coefficient m from Table 1 or, for in-between values, from Diagram 2 using (= 1.135).

Symbols and units

- p_1 } Absolute pressure
 p_2 } in bar
 Δp Differential pressure in bar
 W Mass flow rate in kg/h
 K_V K_V value in m³/h
 m Head loss coefficient, dimensionless
 Z Compressibility factor, dimensionless

Sizing sample problem 5

- Determine:** Steam mass flow rate in kg/h with valve fully open
- Given:** Type 3241-2 Electric Control Valve · Steam temperature · Upstream and downstream pressures
- Solution:** Calculate $\frac{p_2}{p_1}$ (convert into absolute pressure if required)
- Determine m from Table 1 or Diagram 2
- Determine Z from Table 2 using the upstream pressure and temperature

$$W = K_{VS} \cdot m \cdot Z \quad (5)$$

- $K_{VS} = 35 \text{ m}^3/\text{h}^{1)}$
 $t = 200 \text{ }^\circ\text{C}$
 $p_1 = 4 \text{ bar}$ $p_2 = 3 \text{ bar}$
 $\frac{p_2}{p_1} = \frac{3}{4} = 0.75$
 $m = 0.92$
 $Z = 38.5$
 $W = 35 \cdot 0.92 \cdot 38.5 = 1240 \text{ kg/h}$

¹⁾ K_{VS} and hence also W, the calculated mass flow rate, have a tolerance of ±10 %.

Sizing sample problem 6

- Determine:** Type ... Pressure Reducing Valve, K_{VS} value, nominal size
- Given:** Steam mass flow rate · Steam temperature · Upstream and downstream pressures
- Solution:** Calculate $\frac{p_2}{p_1}$ (convert into absolute pressure if required)
- Determine m from Table 1 · Determine Z from Table 2 using the upstream pressure and temperature
- Calculate K_V from Equation (6):

$$K_V = \frac{W}{Z \cdot m} \quad (6)$$

- Determine the K_{VS} value of the valve from the following equation using the calculated K_V value.
- In general: $K_{VS} \approx 1.3 \cdot K_V$

- $W = 1000 \text{ kg/h}$
 $t =$ corresponds to saturated steam
 $p_1 = 7 \text{ bar}$ $p_2 = 2 \text{ bar}$
 $\frac{p_2}{p_1} = \frac{2}{7} = 0.286$
 $m = 1$
 $Z = 71.3$

$$K_V = \frac{1000}{71.3 \cdot 1} = 14 \text{ m}^3/\text{h}$$

$$K_{VS} = 1.3 \cdot K_V = 1.3 \cdot 14 = 17.5 \text{ m}^3/\text{h}$$

Selection: Type 39-2, DN 40, $K_{VS} = 20$

Sizing sample problem 7

- Determine:** Differential pressure $\Delta p = p_1 - p_2$ across fully open valve
- Given:** Type 4 Self-operated Temperature Regulator · Steam mass flow rate · Steam temperature · Upstream pressure
- Solution:** Determine Z from Table 2 using the upstream pressure and temperature

$$m = \frac{W}{Z \cdot K_{VS}} \quad (7)$$

- Using $m = 0.701$, determine the ratio of $\frac{p_2}{p_1}$ from Diagram 2
- $p_2 = 0.884 \cdot p_1 \Rightarrow$ Differential pressure $\Delta p = p_1 - p_2$

- $K_{VS} = 20 \text{ m}^3/\text{h}$
 $W = 1000 \text{ kg/h}$
 $t =$ corresponds to saturated steam
 $p_1 = 7 \text{ bar}$
 $Z = 71.3$
 $m = \frac{1000}{71.3 \cdot 20} = 0.701$
 $\frac{p_2}{p_1} = 0.884$
 $p_2 = 0.884 \cdot 7 = 6.2 \text{ bar}$
 $\Delta p = p_1 - p_2 = 7 - 6.2 = 0.8 \text{ bar}$

Table 1 · Head loss coefficient m as a function of p_2/p_1

Pressure ratio p_2/p_1	0 to 0.6	0.70	0.75	0.80	0.85	0.90	0.95	0.99
Head loss coefficient m	1.0	0.96	0.92	0.86	0.77	0.66	0.48	0.22

Table 2 · Compressibility factor Z for steam · All pressures are indicated as absolute pressures in bar

Compressibility factor Z for ...														
p_1 in bar	Saturated steam	Superheated steam at the following temperatures ...												
		60 °C	80 °C	100 °C	120 °C	140 °C	160 °C	180 °C	200 °C	250 °C	300 °C	350 °C	400 °C	
0.1	1.16	1.13	1.1	1.07	1.04	1.02	0.99	0.97	0.95	0.90	0.86	0.83	0.80	
0.2	2.27	2.27	2.21	2.15	2.09	2.04	1.99	1.95	1.90	1.81	1.73	1.66	1.59	
0.3	3.37		3.31	3.22	3.14	3.06	2.99	2.92	2.86	2.71	2.59	2.49	2.39	
0.4	4.45		4.42	4.29	4.18	4.08	3.98	3.89	3.81	3.62	3.46	3.32	3.19	
0.5	5.53			5.37	5.23	5.10	4.98	4.86	4.76	4.52	4.33	4.15	3.99	
0.6	6.58			6.45	6.28	6.12	5.97	5.84	5.72	5.43	5.19	4.98	4.78	
0.7	7.65			7.53	7.33	7.15	6.97	6.82	6.67	6.34	6.06	5.80	5.59	
0.8	8.71			8.62	8.39	8.17	7.97	7.79	7.63	7.25	6.91	6.64	6.37	
0.9	9.76			9.70	9.44	9.19	8.98	8.77	8.58	8.16	7.90	7.37	7.18	
1.0	10.8			10.8	10.5	10.2	9.98	9.76	9.53	9.07	8.66	8.30	7.98	
1.1	11.9				11.5	11.3	11.0	10.8	10.5	10.0	9.50	9.10	8.70	
1.2	12.9				12.6	12.3	12.0	11.8	11.4	10.9	10.4	10.0	9.60	
1.3	13.9				13.7	13.3	13.0	12.7	12.3	11.8	11.2	10.8	10.4	
1.4	15.0				14.7	14.3	14.0	13.7	13.4	12.7	12.1	11.6	11.2	
1.5	16.0				15.8	15.4	15.0	14.7	14.3	13.6	13.0	12.4	12.0	
1.6	17.0				16.9	16.4	16.0	15.6	15.3	14.5	13.9	13.3	12.8	
1.7	18.0				17.9	17.5	17.0	16.6	16.3	15.4	14.7	14.1	13.6	
1.8	19.1				19.0	18.5	18.0	17.6	17.2	16.4	15.6	14.9	14.4	
1.9	20.1				20.1	19.5	19.0	18.6	18.1	17.3	16.5	15.8	15.2	
2.0	21.1				21.1	20.6	20.0	19.6	19.1	18.2	17.3	16.6	16.1	
2.2	23.2					22.6	22.1	21.5	21.0	20.0	19.1	18.3	17.6	
2.4	25.2					24.7	24.1	23.5	23.1	21.8	20.8	20.0	19.2	
2.6	27.2					26.8	26.0	25.5	24.9	23.6	22.6	21.5	20.8	
2.8	29.3					28.9	28.1	27.5	26.8	25.5	24.3	23.2	22.4	
3.0	31.0					31.0	30.2	29.4	28.8	27.3	26.0	24.9	24.0	
3.2	33.4					33.1	32.2	31.4	30.7	29.1	27.8	26.6	25.6	
3.4	35.4					35.2	34.3	33.4	32.6	31.0	29.6	28.2	27.2	
3.6	37.4					37.3	36.3	35.4	34.6	32.8	31.3	29.9	28.9	
3.8	39.4						38.3	37.4	36.5	34.7	33.0	31.6	30.4	
4.0	41.4						40.4	39.4	38.5	36.5	35.1	33.3	32.0	
4.5	46.4						45.6	44.4	42.8	41.1	39.1	37.3	36.1	
5.0	51.4						50.8	49.4	48.2	45.7	43.6	41.8	40.0	
5.5	56.4						56.0	54.4	53.0	50.2	47.8	46.7	44.2	
6.0	61.4						61.2	59.5	57.9	54.9	52.3	50.2	48.2	
6.5	66.3							64.6	62.9	59.4	56.6	54.2	52.2	
7.0	71.3							69.7	67.8	64.2	61.1	58.3	56.2	
8.0	81.2							79.9	77.6	73.4	69.8	67.0	64.3	
9.0	91.0							90.2	87.7	82.6	78.7	75.0	72.4	
10.0	101							101	97.9	92.2	87.4	83.2	80.4	
11.0	111								108	102	96.5	92.1	88.5	
12.0	121								118	111	105	99.7	96.7	
13.0	130								128	121	114	109	105	
14.0	140									139	130	123	118	113
15.0	150									150	139	132	125	121
16.0	160										149	141	134	129
17.0	170										159	150	143	137
18.0	180										169	159	151	146
19.0	189										178	168	161	154
20.0	199										188	177	168	162
21.0	209										198	187	178	170
23.0	229										218	205	195	187
25.0	248										238	224	213	203
27.0	268										258	242	230	216
29.0	288										279	261	248	236
31.0	308										300	280	264	253
33.0	328										322	299	282	270
35.0	348										343	318	301	286
37.0	368										365	338	319	304
39.0	388										387	356	337	320
41.0	408											376	354	338

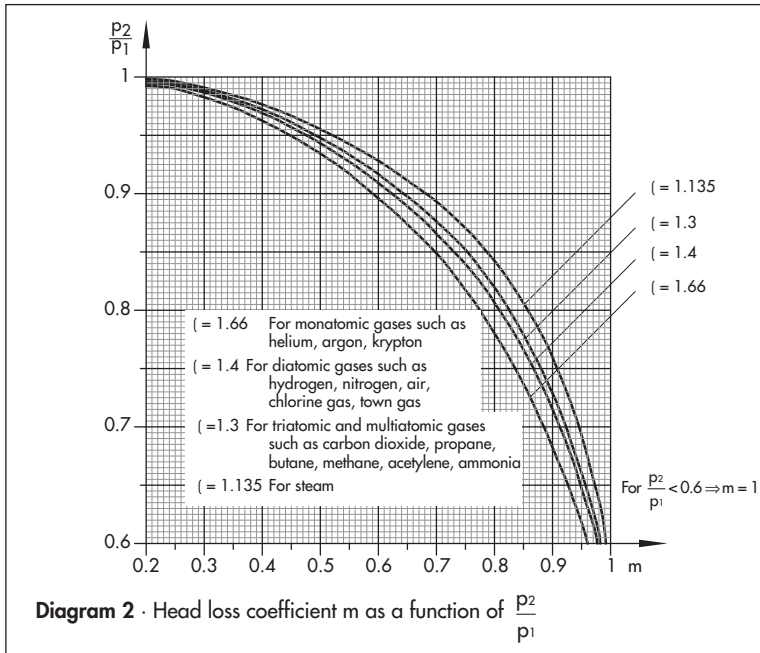


Diagram 2 · Head loss coefficient m as a function of $\frac{p_2}{p_1}$

Symbols and units

- p_1 } Absolute pressure
- p_2 } in bar
- Δp Differential pressure in bar
- W Mass flow rate in kg/h
- K_V K_V value in m^3/h
- ρ_1 Density in kg/m^3

The calculation procedure provides approximate values meeting practical needs.

$$W = 14.2 \cdot K_V \cdot m \cdot \sqrt{p_1 \cdot \rho_1} \quad (8)$$

The values for m and ρ_1 can be found in Diagrams 2 and 3 respectively.

Sizing sample problem 8

Determine: Propane gas flow rate W in kg/h with valve fully open
Given: Type 3241-1 Pneumatic Control Valve, DN 50 · Upstream and downstream pressures
Solution: Determine the upstream density ρ_1 from Diagram 3
 Calculate $\frac{p_2}{p_1}$. Using $\frac{p_2}{p_1}$ and $(\zeta = 1.3)$, determine m from Diagram 2.

$K_{VS} = 35 \text{ m}^3/h$ ¹⁾
 $p_1 = 2.7 \text{ bar}$ $p_2 = 2.2 \text{ bar}$
 $\rho_1 = 5.28 \text{ kg/m}^3$
 $\frac{p_2}{p_1} = \frac{2.2}{2.7} = 0.815$
 $m = 0.805$
 $W = 14.2 \cdot 35 \cdot 0.805 \cdot \sqrt{2.7 \cdot 5.28} = 1511 \text{ kg/h}$

$$W = 14.2 \cdot K_{VS} \cdot m \cdot \sqrt{p_1 \cdot \rho_1} \quad (8)$$

¹⁾ K_{VS} and hence also W , the calculated mass flow rate, have a tolerance of $\pm 10\%$.

Sizing sample problem 9

Determine: Type ... Pressure Reducing Valve, K_{VS} value, nominal size
Given: Nitrogen system · Nitrogen flow rate · Pressures p_1 and p_2
Solution: Calculate $\frac{p_2}{p_1}$. Determine m from Diagram 2
 $(\zeta = 1.4)$ · Determine ρ_1 from Diagram 3 using $p_1 = 5 \text{ bar}$

$p_1 = 5 \text{ bar}$ $p_2 = 3 \text{ bar}$
 $W = 250 \text{ kg/h}$
 $\frac{p_2}{p_1} = \frac{3}{5} = 0.6$
 $m = 0.99$
 $\rho_1 = 6.2 \text{ kg/m}^3$
 $K_V = \frac{250}{14.2 \cdot 0.99 \cdot \sqrt{5 \cdot 6.2}} = 3.19 \text{ m}^3/h$
 $K_{VS} = 1.3 \cdot K_V = 1.3 \cdot 3.19 = 3.99 \text{ m}^3/h$

$$K_V = \frac{W}{14.2 \cdot m \cdot \sqrt{p_1 \cdot \rho_1}} \quad (9)$$

Determine the K_{VS} value from the following equation using the calculated K_V value. In general: $K_{VS} \approx 1.3 \cdot K_V$

Selection: Type 44-1 Pressure Regulator, $G \frac{3}{4}$, $K_{VS} = 4$

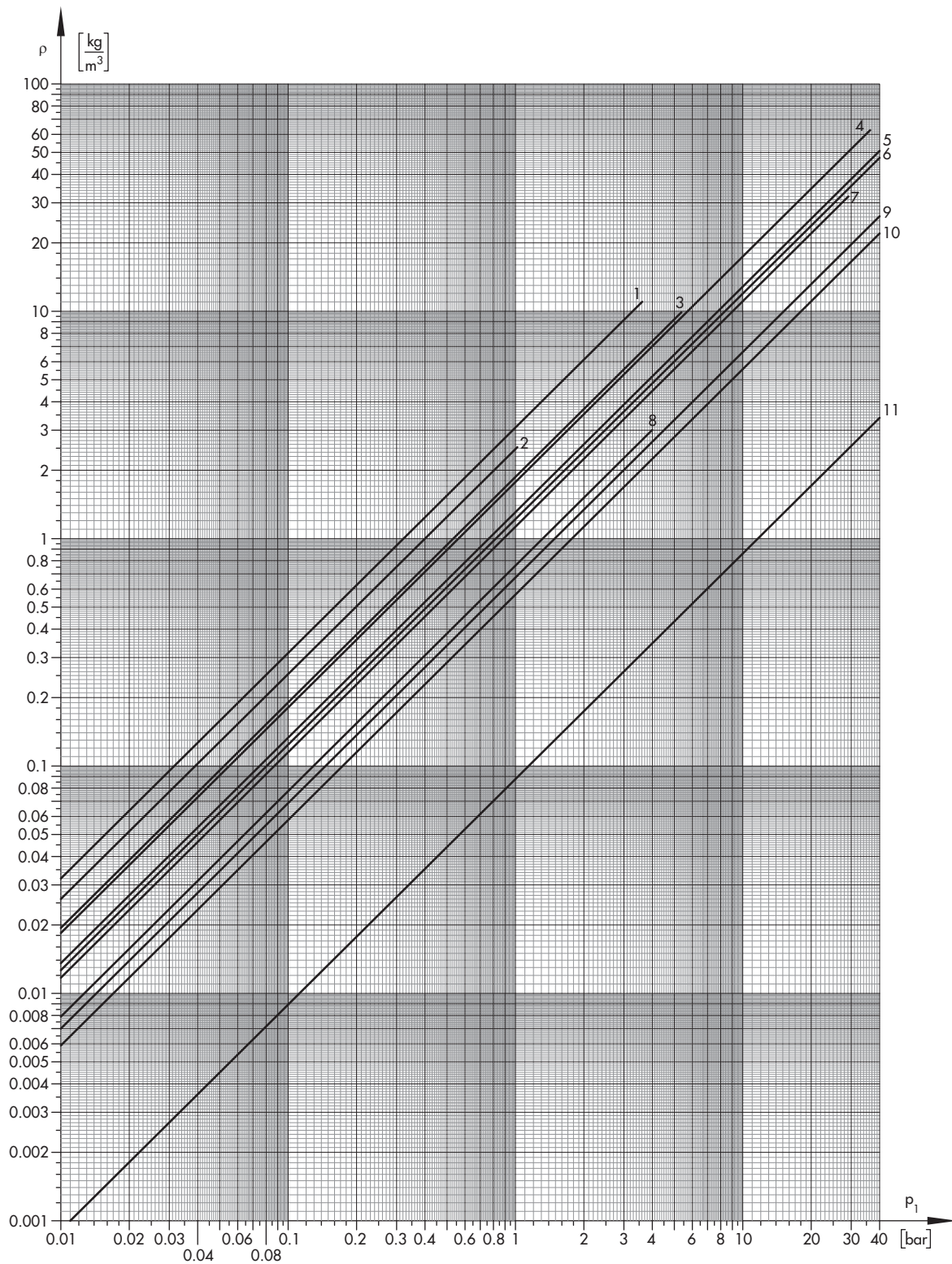
Sizing sample problem 10

Determine: Differential pressure $\Delta p = p_1 - p_2$ across fully open valve
Given: Type 241-2 Electric Control Valve, DN 20 · Nitrogen system · Nitrogen flow rate · Upstream pressure
Solution: Determine ρ_1 from Diagram 3 using $p_1 = 5 \text{ bar}$

$K_{VS} = 4 \text{ m}^3/h$
 $p_1 = 5 \text{ bar}$
 $W = 250 \text{ kg/h}$
 $\rho_1 = 6.2 \text{ kg/m}^3$
 $m = \frac{250}{14.2 \cdot 4 \cdot \sqrt{5 \cdot 6.2}} = 0.791$
 $\frac{p_2}{p_1} = 0.83$
 $p_2 = 0.83 \cdot 5 = 4.15 \text{ bar}$
 $\Delta p = 5 - 4.15 = 0.85 \text{ bar}$

$$m = \frac{W}{14.2 \cdot K_{VS} \cdot \sqrt{p_1 \cdot \rho_1}} \quad (10)$$

Use m to determine $\frac{p_2}{p_1}$ from Diagram 2 $p_2 = 0.83 \cdot p_1$
 $\Delta p = p_1 - p_2$



If the operating temperature t deviates considerably from 0 °C ,
 correct p_1 using the equation $p_t = p_1 \cdot \frac{273}{273+t}$

- | | | |
|------------------|-------------|-------------|
| 1 Chlorine gas | 5 Air | 9 Methane |
| 2 Butane | 6 Nitrogen | 10 Town gas |
| 3 Propane | 7 Acetylene | 11 Hydrogen |
| 4 Carbon dioxide | 8 Ammonia | |

Diagram 3 · Density of gases ρ or p_1 as a function of pressure at 0 °C

The calculation procedure provides approximate values meeting practical needs. The equation derived for dry air is as follows:

$$W = 15.3 \cdot m \cdot K_V \cdot \sqrt{p_1 \cdot \rho_1} \quad (11)$$

Equation (10) can be written as follows:

$$W = K_V \cdot m \cdot Z \quad (12)$$

The value for Z can be obtained from Table 4 using p_1 . The value for m can be found in Table 3 (in-between values can be determined from Diagram 2 using $\lambda = 1.4$)

Symbols and units

p_1	} Absolute pressure
p_2	
Δp	Differential pressure in bar
W	Mass flow rate in kg/h
K_V	K_V value in m^3/h
ρ	Density in kg/m^3
m	Head loss coefficient, dimensionless
Z	Compressibility factor, dimensionless

Note:

Please contact SAMSON AG in Frankfurt/Main to obtain more detailed information on sizing flow regulators for air service.

Sizing sample problem 11

Determine: Mass flow rate in kg/h with valve fully open
 Given: Type 41-23 Self-operated Pressure Regulator, DN 50 · Upstream and downstream pressures · Temperature
 Solution: Calculate $\frac{p_2}{p_1}$ (convert into absolute pressure if required)
 Determine the value for m from Table 3 or Diagram 2
 Determine the value for Z from Table 4 using the upstream pressure and temperature

$$W = K_{VS} \cdot m \cdot Z \quad (12)$$

$K_{VS} = 32 \text{ m}^3/h^{1)}$
 $t = 20 \text{ }^\circ\text{C}$
 $p_1 = 4 \text{ bar } p_2 = 3 \text{ bar}$
 $\frac{p_2}{p_1} = \frac{3}{4} = 0.75$
 $m = 0.884$
 $Z = 66$
 $W = 32 \cdot 0.884 \cdot 66 = 1867 \text{ kg/h}$

¹⁾ K_{VS} and hence also W, the calculated mass flow rate, have a tolerance of $\pm 10 \%$.

Sizing sample problem 12

Determine: Type ... Pressure Reducing Valve, K_{VS} value, nominal size
 Given: Upstream and downstream pressures · Temperature · Process medium is compressed air
 Solution: Calculate $\frac{p_2}{p_1}$ (convert into absolute pressure if required)
 Determine the value for m from Table 3 or Diagram 2 ·
 Determine the value for Z from Table 4 using the upstream pressure and temperature

$$K_V = \frac{W}{Z \cdot m} \quad (13)$$

Determine the K_{VS} value from the following equation using the calculated K_V value. In general: $K_{VS} \approx 1.3 \cdot K_V$

$p_1 = 5 \text{ bar } p_2 = 3 \text{ bar}$
 $t = 20 \text{ }^\circ\text{C}$
 $W = 190 \text{ kg/h}$
 $\frac{p_2}{p_1} = \frac{3}{5} = 0.6$
 $m = 0.982$
 $Z = 82.60$
 $K_V = \frac{190}{0.982 \cdot 82.60} = 2.34 \text{ m}^3/h$
 $K_{VS} = 1.3 \cdot K_V = 1.3 \cdot 2.34 = 2.93 \text{ m}^3/h$
 Selection: Type 44-1, G 1/2, $K_{VS} = 3.2$

Sizing sample problem 13

Determine: Differential pressure $\Delta p = p_1 - p_2$ across fully open valve
 Given: Type 42-24 Differential Pressure Regulator, DN 50 · Flow rate of compressed air · Upstream pressure · Temperature
 Solution: Determine the value for Z from Table 4 using the upstream pressure and temperature

$$m = \frac{W}{Z \cdot K_{VS}} \quad (14)$$

Determine $\frac{p_2}{p_1}$ for $m = 0.884$ from Diagram 2 or Table 3
 $p_2 = 0.75 \cdot p_1 \Rightarrow$ Differential pressure $\Delta p = p_1 - p_2$

$K_{VS} = 32 \text{ m}^3/h$
 $W = 3270 \text{ kg/h}$
 $t = 20 \text{ }^\circ\text{C}$
 $p_1 = 7 \text{ bar}$
 $Z = 115.6$
 $m = \frac{3270}{115.6 \cdot 32} = 0.884$
 $\frac{p_2}{p_1} = 0.75$
 $p_2 = 0.75 \cdot 7 = 5.25 \text{ bar}$
 $\Delta p = p_1 - p_2 = 7 - 5.25 = 1.75 \text{ bar}$

Table 3 · Head loss coefficient m as a function of p_2/p_1 · All indicated pressures are absolute pressures in bar

Pressure ratio p_2/p_1	0.527	0.60	0.65	0.70	0.75	0.80	0.85	0.90	0.95	0.99
Head loss coefficient m	1	0.982	0.978	0.935	0.884	0.818	0.730	0.623	0.448	0.207

Table 4 · Compressibility factor Z for air · All pressures are indicated as absolute pressures in bar

Compressibility factor Z for ...									
p_1 in bar	Dry air at the following temperatures ...								
	0 °C	20 °C	50 °C	100 °C	150 °C	200 °C	250 °C	300 °C	
0.1	1.71	1.65	1.57	1.47	1.38	1.30	1.24	1.18	
0.2	3.42	3.30	3.15	2.93	2.77	2.60	2.47	2.31	
0.3	5.13	4.96	4.74	4.39	4.13	3.89	3.71	3.55	
0.4	6.84	6.61	6.29	5.85	5.50	5.20	4.95	4.72	
0.5	8.55	8.26	7.87	7.32	6.88	6.50	6.18	5.92	
0.6	10.26	9.90	9.42	8.79	8.24	7.79	7.42	7.09	
0.7	11.97	11.56	11.00	10.22	9.61	9.09	8.64	8.27	
0.8	13.68	13.22	12.58	11.72	11.00	10.40	9.79	9.45	
0.9	15.40	14.86	14.15	13.18	12.36	11.69	11.12	10.62	
1.0	17.10	16.50	15.72	14.65	13.75	13.00	12.35	11.81	
1.1	18.83	18.15	17.30	16.07	15.10	14.32	13.60	13.00	
1.2	20.50	19.80	18.20	17.52	16.50	15.60	14.70	14.12	
1.3	22.10	21.42	20.45	18.25	17.85	16.90	16.09	15.35	
1.4	24.00	23.10	22.00	10.70	19.25	18.19	17.30	16.55	
1.5	25.65	24.75	23.60	21.68	20.06	19.46	18.55	17.70	
1.6	27.30	26.40	25.15	23.35	21.95	20.80	19.78	18.88	
1.7	29.10	28.10	26.70	24.80	23.40	22.05	20.90	20.10	
1.8	30.80	29.70	28.30	26.35	24.75	23.35	22.25	21.20	
1.9	32.25	30.25	29.60	27.55	26.15	24.65	23.50	22.50	
2.0	34.20	33.00	31.45	29.27	27.70	25.95	24.70	23.10	
2.2	37.65	36.40	34.70	32.20	30.20	28.50	27.20	25.90	
2.4	41.10	39.60	37.75	35.15	33.30	31.20	29.70	28.35	
2.5	42.70	41.40	39.30	36.60	34.40	32.45	30.90	29.50	
2.6	44.50	42.60	40.90	38.05	35.75	33.80	32.15	30.70	
2.8	47.80	46.20	44.00	41.70	38.45	36.35	34.55	33.10	
3.0	51.30	49.55	47.40	43.95	41.25	38.90	37.10	35.45	
3.2	54.30	52.40	49.90	46.60	43.90	41.60	39.80	37.70	
3.4	58.25	56.20	53.50	49.80	46.70	44.20	42.00	40.10	
3.6	62.20	59.60	56.60	52.70	49.40	46.80	44.50	42.60	
3.8	65.00	62.70	59.75	55.60	51.50	49.40	46.90	44.80	
4.0	68.20	66.00	62.90	58.55	55.00	52.00	49.40	47.20	
4.5	77.00	74.40	70.70	65.80	61.80	58.50	55.60	48.50	
5.0	86.90	82.60	78.75	73.20	68.75	65.00	61.75	59.20	
5.5	94.00	90.90	87.40	80.60	75.60	71.60	68.00	64.90	
6.0	102.06	98.90	94.30	87.90	82.40	77.90	74.15	70.90	
6.5	111.0	107.2	101.10	95.20	88.40	84.50	80.40	76.80	
7.0	119.6	115.6	110.0	102.2	96.90	90.90	86.40	82.70	
8.0	136.8	132.2	125.7	117.1	110.0	104.0	97.9	94.5	
9.0	162.2	148.6	141.6	131.8	123.6	116.9	111.2	106.2	
10.0	171.0	165.0	157.2	146.5	137.5	130.0	123.5	118.1	
11.0	188.3	181.5	173.0	160.7	151.0	143.2	136.0	130.0	
12.0	205.0	198.0	182.0	175.2	165.0	156.0	147.0	141.0	
13.0	221.0	214.2	204.5	182.5	178.5	169.0	160.9	153.5	
14.0	240.0	231.0	220.0	197.0	192.5	181.9	173.0	165.5	
15.0	256.5	247.5	236.0	216.8	200.6	194.6	185.5	177.0	
16.0	273.0	264.0	251.0	253.5	219.5	208.0	197.8	188.8	
17.0	291.0	281.0	267.0	248.0	234.0	220.5	209.0	201.0	
18.0	308.0	297.0	283.0	263.5	247.5	233.5	222.5	212.0	
19.0	322.5	302.5	296.0	275.6	261.6	246.5	235.0	225.0	
20.0	342.0	330.0	314.5	292.7	277.0	259.6	247.0	231.0	
22.0	376.5	364.0	347.0	322.0	302.0	285.0	272.0	259.0	
24.0	411.0	395.0	377.5	351.5	333.0	312.0	297.0	283.5	
26.0	445.0	428.0	409.0	380.5	357.5	338.5	321.5	307.0	
28.0	478.0	462.0	440.0	417.0	384.5	363.5	345.5	331.0	
30.0	513.0	495.5	474.0	439.5	412.5	389.0	371.0	354.5	

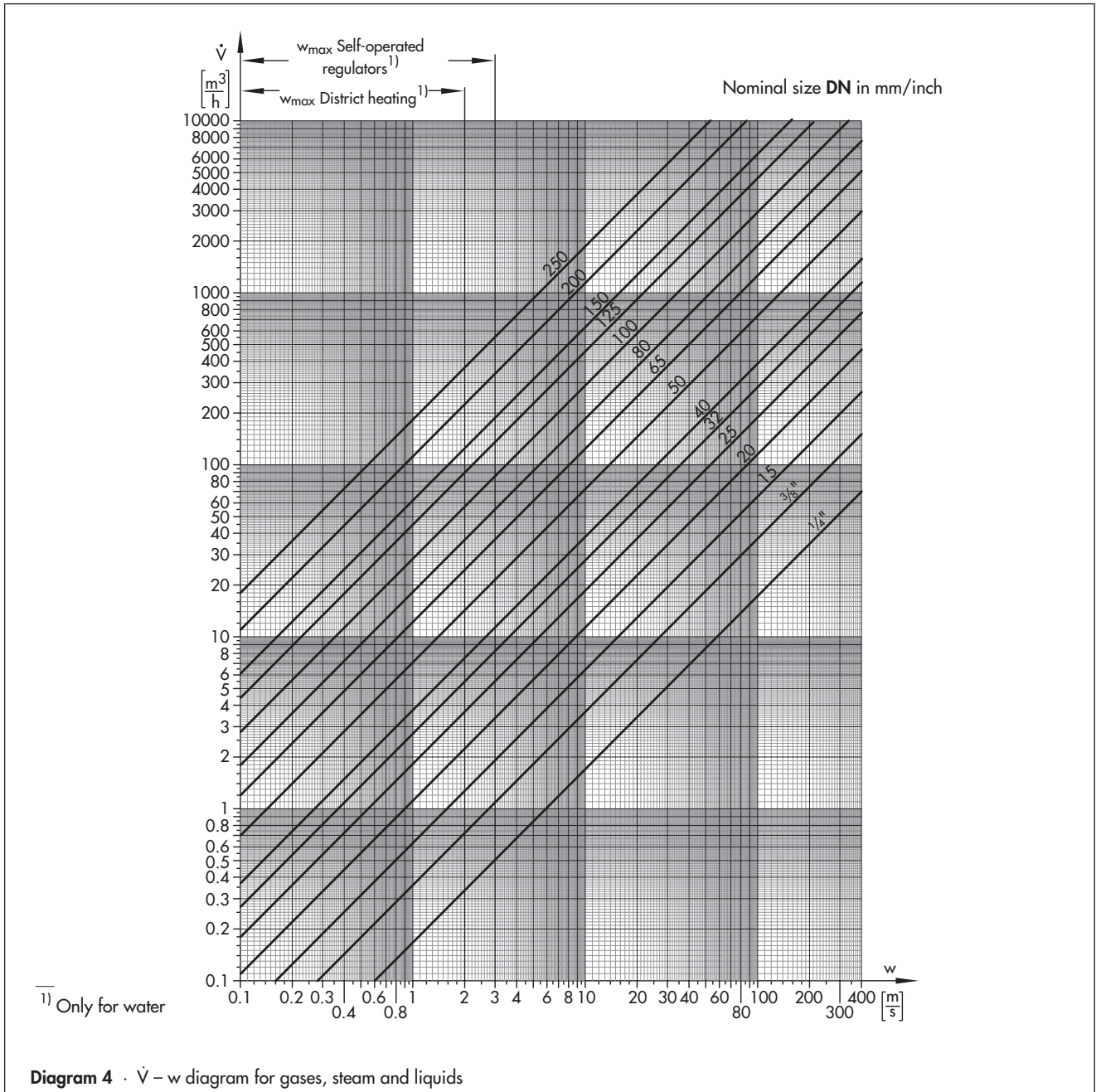


Diagram 4 shows the following relationship:

$$\dot{V}_{\text{pipe}} = A_{\text{pipe}} \cdot w_{\text{pipe}} \quad (15)$$

- \dot{V}_{pipe} Volume flow rate in m³/h
- w_{pipe} Flow velocity in m/s
- A_{pipe} Cross-sectional area of the assumed nominal size plotted as a straight line
- DN Nominal size

Determining velocity and nominal size

Velocity w of the medium

$$w_{\text{pipe}} = \dot{V}_{\text{pipe}} \cdot \left(\frac{18.8}{\text{DN}} \right)^2 \quad (16)$$

For gas service, the flow rate determined from Diagram 4 in m³/h can be converted into kg/h or m³/h for the normal condition (refer to sizing sample problem 14).

Nominal size DN

$$\text{DN} = 18.8 \cdot \sqrt{\frac{\dot{V}}{w_{\text{pipe}}}} \quad (17)$$

- \dot{V}_G Gas flow rate, based on normal condition

Sizing sample problem 14

Determine: Flow rates of compressed air for operating and normal conditions
Given: Pipe diameter · Pressure p in the pipe · Flow velocity
Solution: Volume flow rate in m^3/h for the operating condition can be obtained from Diagram 4 using DN 32 and w_{pipe} . Determine ρ from Diagram 3 using p . Calculate the mass flow rate of the compressed air in kg/h .

$$W = \dot{V} \cdot \rho \quad (18)$$

\dot{V}_G is the volume flow rate of gas in m^3/h for normal condition at 0°C and 1013 mbar . Determine \dot{V}_G according to the equation to the right.

Nominal size = DN 32
 p = 5 bar
 w_{pipe} = 7 m/s
 \dot{V} = 20 m^3/h

From Diagram 3:

ρ = 6.3 kg/m^3
 W = $20 \cdot 6.3 = 126\text{ kg}/\text{h}$

Normal condition:

1 m^3 of air \Rightarrow 1.293 kg

$$\dot{V}_G = \frac{W}{1.293} = \frac{126}{1.293} = 97.5\text{ m}^3/\text{h}$$

Specifications subject to change without notice.



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